### **TM-CLASS CAVITY DESIGN**

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### 500 MHz, Single-cell







### 350 MHz, 4-cell, Nb on Cu







### 1500 MHz, 5-cell









### 1300 MHz 9-cell







### **Pill Box Cavity**





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### Modes in Pill Box Cavity

- TM<sub>010</sub>
  - Electric field is purely longitudinal
  - Electric and magnetic fields have no angular dependence
  - Frequency depends only on radius, independent on length
- TM<sub>0mn</sub>
  - Monopoles modes that can couple to the beam and exchange energy
- **TM**<sub>1mn</sub>
  - Dipole modes that can deflect the beam
- TE modes
  - No longitudinal E field
  - Cannot couple to the beam





### TM Modes in a Pill Box Cavity

$$\frac{E_r}{E_0} = -\frac{n\pi}{x_{lm}} \frac{R}{L} J_l' \left( x_{lm} \frac{r}{R} \right) \sin\left(n\pi \frac{z}{L}\right) \cos l\varphi$$

$$\frac{E_{\varphi}}{E_0} = \frac{ln\pi}{x_{lm}^2} \frac{R^2}{rL} J_l \left( x_{lm} \frac{r}{R} \right) \sin\left(n\pi \frac{z}{L}\right) \sin l\varphi$$

$$\frac{E_z}{E_0} = J_l \left( x_{lm} \frac{r}{R} \right) \sin\left(n\pi \frac{z}{L}\right) \cos l\varphi$$

$$\omega_{lmn} = c_{\sqrt{\left(\frac{x_{lm}}{R}\right)^2 + \left(\frac{\pi n}{L}\right)^2}}$$

$$\frac{H_r}{E_0} = -i\omega\varepsilon \frac{l}{x_{lm}^2} \frac{R^2}{r} J_l \left( x_{lm} \frac{r}{R} \right) \cos\left( n\pi \frac{z}{L} \right) \sin l\varphi$$
$$\frac{H_{\varphi}}{E_0} = -i\omega\varepsilon \frac{R}{x_{lm}} J_l' \left( x_{lm} \frac{r}{R} \right) \cos\left( n\pi \frac{z}{L} \right) \cos l\varphi$$
$$\frac{H_z}{E_0} = 0$$

$$x_{lm}$$
 is the mth root of  $J_l(x)$ 





### **TM<sub>010</sub> Mode in a Pill Box Cavity**

$$E_r = E_{\varphi} = 0 \qquad \qquad E_z = E_0 J_0 \left( x_{01} \frac{r}{R} \right)$$
$$H_r = H_z = 0 \qquad \qquad H_{\varphi} = -i\omega \varepsilon E_0 \frac{R}{x_{01}} J_1 \left( x_{01} \frac{r}{R} \right)$$

$$\omega = x_{01} \frac{c}{R}$$
  $x_{01} = 2.405$ 

$$R = \frac{x_{01}}{2\pi}\lambda = 0.383\lambda$$





### **TM<sub>010</sub> Mode in a Pill Box Cavity**

Energy content

$$U = \varepsilon_0 E_0^2 \frac{\pi}{2} J_1^2(x_{01}) LR^2$$

Power dissipation x $P = E_0^2 \frac{R_s}{\eta^2} \pi J_1^2(x_{01})(R+L)R \qquad J$ 

$$x_{01} = 2.40483$$
$$J_1(x_{01}) = 0.51915$$

Geometrical factor

$$G = \eta \frac{x_{01}}{2} \frac{L}{(R+L)}$$





### TM010 Mode in a Pill Box Cavity

**Energy Gain** 

$$\Delta W = E_0 \frac{\lambda}{\pi} \sin \frac{\pi L}{\lambda}$$

Gradient

$$E_{acc} = \frac{\Delta W}{\lambda/2} = E_0 \frac{2}{\pi} \sin \frac{\pi L}{\lambda}$$

Shunt impedance

$$R_{sh} = \frac{\eta^2}{R_s} \frac{1}{\pi^3 J_1^2(x_{01})} \frac{\lambda^2}{R(R+L)} \sin^2\left(\frac{\pi L}{\lambda}\right)$$





### **Real Cavities**

Beam tubes reduce the electric field on axis







### **Real Cavities**







### **Single Cell Cavities**







### **Single Cell Cavities**







### **Multi-Cell Cavities**











### **Multi-Cell Cavities**



: Sketch of the electric field lines of the  $\pi$ -mode of a 5-cell :





### **Cell-to-cell Coupling**

0 mode

 $\mathcal{W}_0$ 

 $\pi \mod \omega_{\pi}$ 



 $k = \frac{2(\omega_{\pi} - \omega_0)}{\omega_{\pi} + \omega_0}$ 





### **Multi-Cell Cavities**



Mode frequencies:

$$\frac{\omega_m^2}{\omega_0^2} = 1 + 2k \left( 1 - \cos \frac{\pi m}{n} \right)$$
$$\frac{\omega_n - \omega_{n-1}}{\omega_0} \approx k \left( 1 - \cos \frac{\pi}{n} \right) \approx \frac{k}{2} \left( \frac{\pi}{n} \right)^2$$

Voltages in cells:

$$V_j^m = \sin\left(\pi m \frac{2j-1}{2n}\right)$$





### **Pass-Band Modes Frequencies**







### **Cell Excitations in Pass-Band Modes**







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### **Field Flatness**

Geometrical differences between cells causes a mixing of the eigenmodes

Sensitivity to mechanical deformation depends on mode spacing







### **Mechanical Design**

The mechanical design of a cavity follows its RF design:

- **Lorentz Force Detuning** •
- **Mechanical Resonances** •



*E* and *H* at  $E_{acc} = 25 \text{ MV/m}$  in TESLA inner-cup





### **Mechanical Design**



 $k_L = -1 \ Hz/(MV/m)^2$ 

Essential for the operation of a pulsed accelerator  $\Delta f = k_L (E_{acc})^2$ 





### **Mechanical Design**

Mechanical Resonances of a multi-cell cavity



TESLA structure

The mechanical resonances modulate frequency of the accelerating mode. Sources of their excitation: vacuum pumps, ground vibrations...





### **Cell Shape Parametrization**







### **Tools used for the parametrization**

Using of the parametric tool developed at *INFN Milano* for the analysis of the cavity shape on the electromagnetic parameters:

- All RF computations are handled by SUPERFISH
- Inner cell tuning is performed through the cell diameter, all the characteristic cell parameters stay constant: R, r, , d, L, Riris
- End cell tuning is performed through the wall angle inclination, , or distance, d.
  - R, L and Riris are independently settable.
- Multicell cavity is then built to minimize the field unflatness, compute the effective and the final cavity performances.
- A proper file to transfer the cavity geometry to ANSYS is then generated



6 cell cavity [Internal Cell: 671 Left Cell: 106 Right Cell: 8 From database: C:\BCavWF\Cav2504.mdb]

beta



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### R: "mechanical" parameter

- The equator aspect ratio (R) is a free parameter for what concerns the -mode e.m. design but the cavity mechanical parameters are greatly affected by the equator shape: R>1 allows better stress distribution in the unstiffened cavity but a bigger Lorentz force coefficient
- The cell tuning strategy doesn't affect the cell's performances (e.m. and mechanical)



Reference data						
L = 56.8 mm r = 1.7 $\alpha = 7^{\circ}$						
$d = 11 \text{ mm}$ $R_{iris} = 43 \text{ mm}$						







### **Optimal value for r**







### **Optimal value for r**



### Influence of d @ constant R<sub>iris</sub>



 Better mechanical performances are reached with decreasing d



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0.8

2.5

2

1.5

-82

1 71

0.9

1.61

d [cm] @ r = b/a optimal

1.53

1.1

1.46

1.2



 $\alpha = 7^{\circ}$ 

30

25

20

10

5

0

139

1.3

15 **g** 

### Influence of d @ constant k

• If we want to keep a constant cell to cell coupling we have to adjust Riris







### Dependence on $\boldsymbol{\alpha}$

The wall angle  $\alpha$  slightly affects all the e.m. parameters, but has a strong effect on the

mechanical performances:

- Lower values are preferred for Lorentz force detuning
- Too small α could be critical for chemistry and cleaning

Chosen  $\alpha = 7 \text{ deg}$ 







### End cell tune



- d set 1 mm lower than the in-cell
- optimization of  $\mathbf{r} = \mathbf{b}/\mathbf{a}$  at iris
- Slater compensation (decrease of the magnetic volume) of the cut-off tube and d reduction (↓f), increasing the wall angle α. This gives also the necessary stiffening to the end cell
- the frequency of end cell + tube is about 50 kHz lower than the in-cell's due to the asimmetry





### End cell @ FPC side tune







### **Optimal stiffening ring position**



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### **KL for different boundary conditions**

- The estimate for KL strongly depends on the cell boundaries. We compute it for 3 different cases:
  - Fixed cell length
  - Free cell length
  - Helium Vessel/Tuning System (= 3 tubes with diameter 30 mm and thickness 2 mm)







# $\beta_g$ = 0.61 Cavity for SNS

Effective  $\beta$  that matches the TTF curve = 0.630

E <sub>p</sub> /E <sub>acc</sub> B <sub>p</sub> /E <sub>acc</sub> [mT/(MV/m)]	2.72 (2.63 inner cell) 5.73 (5.44 inner cell)	
R/Q [Ω]	279	
<b>G</b> [Ω]	214 $\bigcirc \bigcirc \bigcirc$	
k [%]	1.53	
Q <sub>BCS</sub> @ 2 K [10 <sup>9</sup> ]	27.8	
Frequency [MHz]	805.000	
Field Flatness [%]	2	

Nb thickness = 3.8 mm-

 $KL70 = -2.9 [Hz/(MV/m)^2]$   $KL80 = -3.4 [Hz/(MV/m)^2]$ 

	Geo	ometrical Parameters		
	Inner cell	End Cell Left	End Grou	ıp (coupler)
			Left	Right
L [cm]	5.68	5.68	5.	68
R <sub>iris</sub> [cm]	4.3	4.3	4.3	6.5
D [cm]	16.376	16.376	16.	698
d [cm]	1.1	1.0	1.1	1.0
r	1.7	1.5	1.7	1.5
R	1.0	1.0	1	.0
$\alpha$ [deg]	7.0	8.36	7.0	10.0





# $\beta_g$ = 0.81 Cavity for SNS

Effective  $\beta$  that matches the TTF curve = 0.832

$E_p/E_{acc}$	2.19 (2.14 inner	cell)		
B <sub>p</sub> /E <sub>acc</sub> [mT/(MV/m)]	4.72 (4.58 inner	cell)		
R/Q [Ω]	484.8		$\bigtriangleup$	
<b>G</b> [Ω]	233			
k [%]	1.52			
Q <sub>BCS</sub> @ 2 K [10 <sup>9</sup> ]	36.2			
Frequency [MHz]	805.004 KI	.70 = -0.7 [Hz/(MV/m	) <sup>2</sup> ] KL80 = ·	-0.8 [Hz/(MV/m) <sup>2</sup> ]
Field Flatness [%]	1.1	Nb thick	kness = 3.8 mm-	
	Geometrie	al Parameters		
Inn	er cell E	End Cell Left	End Group	(coupler)
			Left	Right
L [cm]	7.55	7.55	7.55	
R <sub>iris</sub> [cm]	4.88	4.88	4.88	7.0
D [cm]	16.415	16.415	<b>16.61</b> 1	
d [cm]	1.5	1.3	1.5	1.3
r	1.8	1.6	1.8	1.6
R	10	10	10	
	1.0		1.0	





### **Stress and Modal Analysis**

• Nominal Medium Beta Cavity







### **SNS Cavity Modal Analysis**

### Medium Beta Cavity

End Condition	Load (atm)	127 mm Rings (Hz)	70 mm Rings (Hz)	No Rings (Hz)
Fixed-Guided	-	85	48	38
Fixed-Fixed	_	126 (*204)	57 (*59)	48 (*42)
Fixed-Fixed Mid Supt	_	149 (*220)	95 (~*108)	88
Compressed 0.4mm	1.65	125	-	46
Compressed 1.25 mm	1.65	124	-	46

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### **SNS Cavity Modal Analysis**

### High Beta Cavity

End Condition	Load	127 mm	70 mm Rings	No Rings
	(atm)	Rings (Hz)	(Hz)	(Hz)
Fixed-Fixed	-	120	-	46
Fixed-Guided	_	107	-	34
Compressed 0.4mm	1.65	120	-	44
Compressed 1.25 mm	1.65	119	-	44





### Mode Analysis, Beta = 0.81





Mode 1 – 14 Hz



Mode 3 – 40 Hz



Mode 2 – 26 Hz



Mode 5 – 72 Hz



	Natural Frequency (Hz)		
Mode	Test Data	FE Analysis	
1	13	14	
2	31	26	
3	38	40	
4	53	48	
5	70	72	
6	82	83	
7	125	124	





### **SNS Cavity Mechanical Design Requirements**

- Minimize/prevent microphonics
- Withstand loss of vacuum accident up to 5 atm
- Withstand cool down at 1.65 atm
- Adhere to intent of ASME B&P Code
  - Allowable Stress (Sm) = 2/3 Yield Stress
  - Primary Membrane Stress (Pm) <= (Sm)</p>
  - Pm + Bending <= 1.5\*Sm</p>
  - Pm + Bending + Secondary Stress <= 3\*Sm</p>
    - Allowable Stresses

# »Warm Niobium = 4,667 psi »Cold Niobium = 53,333 psi





### **Medium Beta Stress Analysis**

SNS Medium Beta Cavity Wall Stresses					
Compression (mm/end)	Loads (atm)	127 mm Stiffening Ring Max Stress (psi)	70 mm Stiffening Ring Max Stress (psi)	No Stiffening Ring Max Stress (psi)	
0.2	1.65	_	-	-	
0.4	1.65	3,960	-	4,310	
0.5	1.65	4,610	-	4,550	
0.75	1.65	7,500	-	4,670	
1.25	1.65	17,500	5,730 (1.8 atm)	5,000	
0.75	5	11,200	-	12,900	
1.25	5	14,300	10,100	47,100	





### **High Beta Stress Analysis**

SNS High Beta Cavity Wall Stresses					
Compression	Loads	127 mm	70 mm Stiffening	No Stiffening	
(mm/end)	(atm)	Stiffening Ring	Ring	Ring	
		Max Stress	Max Stress	Max Stress	
		(psi)	(psi)	(psi)	
0.2	1.65	3,040	-	-	
0.4	1.65	6,350	-	3,140	
0.5	1.65	8,070	-	3,350	
0.75	1.65	12,500	-	3,940	
1.25	1.65	21,400	-	5,830	
0.75	5	11,500	-	9,130	
1.25	5	14,300	-	9,590	



